AMENDMENTS TO THE CLAIMS

The following listing of claims will replace all prior versions and listings of claims in the application:

1. (currently amended) An eccentric transmission, comprising:

an imbalance compensation element (10a – 10e);

an eccentric element (12a – 12e);

at least one ball bearing which is coupled to the eccentric element (12a – 12e);

an armature shaft (14a – 14e) having a rotation axis;

an oscillating link (32a – 32e); and

a drive shaft (16a – 16e),

wherein a center of mass of a total system comprising the eccentric element (12a – 12e) and the at least one ball bearing lies on the rotation axis,

wherein the eccentric element (12a – 12e) has an armature recess receiving the armature shaft (14a – 14e), is rotatably and fixedly mounted on the armature shaft (14a – 14e) at the armature recess, rotates with the armature shaft (14a – 14e) and converts, due to its own rotation during an operation mode, a revolving rotary motion of the armature shaft (14a – 14e) into an oscillating rotary motion of the drive shaft (16a – 16e) via the oscillating link (31a – 32e) in order to drive an insertion tool (40a – 40e) of a hand-held power tool (18a – 18e)

to oscillate, wherein the imbalance compensation element (10a – 10e) is a one-piece part of an additional functional unit (12a – 12d, 14e).

2. (original) The eccentric transmission as recited in claim 1, wherein the additional functional unit is the eccentric element (12a – 12d).

3. (previously presented) The eccentric transmission as recited in claim 1, wherein the imbalance compensation element (10a, 10d, 10e) includes a recess.

4. (previously presented) The eccentric transmission as recited in claim 1, wherein the imbalance compensation element (10b, 10c) is composed of an outer casing (22b, 22c) of the eccentric element (12b, 12c).

5. (original) The eccentric transmission as recited in claim 4, wherein an axis (20c) of the outer casing (22c) is tilted in relation to at least one axis (24c, 26c) of the eccentric element (12c).

6. (previously presented) The eccentric transmission as recited in claim 1, wherein the eccentric element (12a – 12e) is provided to be press-fitted onto the armature shaft (14a – 14e).

7. (previously presented) The eccentric transmission as recited in claim 1, wherein the imbalance compensation element (10d) has a cross section that changes in the axial direction.

8. (previously presented) The eccentric transmission as recited in claim 1, wherein the imbalance compensation element (10d) has at least two axially offset regions (28d, 30d), each with a different imbalance.

9. (previously presented) The eccentric transmission as recited in claim 1, wherein the additional functional unit is the armature shaft (14e) of an electric motor (36e).

10. (original) The eccentric transmission as recited in claim 9, wherein the imbalance compensation element (10e) includes a recess in the armature shaft (14e).

11. (original) The eccentric transmission as recited in claim 10, wherein the imbalance compensation element (10e) includes a lateral flattened region of the armature shaft (14e).

12. (previously presented) A hand-held power tool equipped with an eccentric transmission as recited in claim 1.

13. (currently amended) An eccentric transmission, comprising:

an imbalance compensation element (10a – 10e);

an eccentric element (12a – 12e);

at least one ball bearing which is coupled to the eccentric element (12a – 12e);

an armature shaft (14a – 14e) having a rotation axis;

an oscillating link (32a – 32e); and

a drive shaft (16a – 16e),

wherein a center of mass of a total system comprising the eccentric element (12a – 12e) and the at least one ball bearing lies on the rotation axis,

wherein the eccentric element (12a – 12e) has an armature recess receiving the armature shaft (14a – 14e), is fixedly mounted on the armature shaft (14a – 14e) at the armature recess, converts in an operation mode a revolving rotary motion of the armature shaft (14a – 14e) into an oscillating rotary motion of the drive shaft (16a – 16e) via the oscillating link (32a – 32e) in order to drive an insertion tool (40a – 40e) of a hand-held power tool (18a – 18e) to oscillate, wherein the imbalance compensation element (10a – 10e) is a one-piece part of an additional functional unit (12a – 12d, 14e), wherein an axis (20c) of the outer casing (22c) is tilted in relation to at least one axis (24c, 26c) of the eccentric element (12c).

14. (currently amended) An eccentric transmission, comprising:

an imbalance compensation element (10a – 10e);

an eccentric element (12a – 12e);

at least one ball bearing which is coupled to the eccentric element (12a – 12e);

an armature shaft (14a – 14e) having a rotation axis;

an oscillating link (32a – 32e); and

a drive shaft (16a – 16e),

wherein a center of mass of a total system comprising the eccentric element (12a – 12e) and the at least one ball bearing lies on the rotation axis,

wherein the eccentric element (12a – 12e) has an armature recess receiving the armature shaft (14a – 14e), is fixedly mounted on the armature shaft (14a – 14e) at the armature recess, converts in an operation mode a revolving rotary motion of the armature shaft (14a – 14e) into an oscillating rotary motion of the drive shaft (16a – 16e) via the oscillating link (32a – 32e) in order to drive an insertion tool (40a – 40e) of a hand-held power tool (18a – 18e) to oscillate, wherein the imbalance compensation element (10a – 10e) is a one-piece part of an additional functional unit (12a – 12d, 14e), wherein the additional functional unit is the armature shaft (14e) of an electric motor (36e).

15. (previously presented) The eccentric transmission as recited in claim 14, wherein the imbalance compensation element (10e) includes a recess in the armature shaft (14e).

16. (previously presented) The eccentric transmission as recited in claim 15, wherein the imbalance compensation element (10e) includes a lateral flattened region of the armature shaft (14e).

17-18. (canceled)

19. (previously presented) The eccentric transmission as recited in claim 1, wherein the eccentric element (12a – 12e) has an opening (48a – 48e), for letting air escape from the recess.

20. (previously presented) The eccentric transmission as recited in claim 1, wherein the eccentric element (12a – 12e) and the armature shaft (14a – 14e) rotate about a same axis.

21. (currently amended) An eccentric transmission, comprising:

an imbalance compensation element (10a – 10e);

a ball bearing (34a – 34e);

an eccentric element (12a – 12e) coupled to the ball bearing (34a – 34e);

an armature shaft (14a – 14e) having a rotation axis;

an oscillating link (32a – 32e); and

a drive shaft (16a – 16e),

wherein a center of mass of a total system comprising the eccentric element (12a – 12e) and the ball bearing lies on the rotation axis,

wherein the eccentric element (12a – 12e) has an armature recess receiving the armature shaft (14a – 14e), is rotatably and fixedly mounted on the armature shaft (14a – 14e) at the armature recess, rotates with the armature shaft (14a – 14e) and converts, due to its own rotation during an operation mode, a revolving rotary motion of the armature shaft (14a – 14e) into an oscillating rotary motion of the drive shaft (16a – 16e) via the oscillating link (31a – 32e) in order to drive an insertion tool (40a – 40e) of a hand-held power tool (18a – 18e) to oscillate,

wherein the imbalance compensation element (10a – 10e) is a one-piece part of an additional functional unit (12a – 12d, 14e),

wherein the oscillating link (32a – 32e) is fork-shaped, rests against both sides of an outer circumference of the ball bearing (34a – 34e) and is non-rotatably connected to the drive shaft (16a – 16e).

22-23. (canceled)

24. (previously presented) The eccentric transmission as recited in claim 1, wherein the armature shaft (14a – 14e) and the drive shaft (16a – 16e) are substantially arranged perpendicular to each other.

25. (previously presented) The eccentric transmission as recited in claim 1, wherein the armature shaft (14a – 14e) and the oscillating link (32a – 32e) are parallel to each other in at least one operation mode of the eccentric transmission.

26. (previously presented) The eccentric transmission as recited in claim 1, wherein the oscillating link (32a – 32e) is fork-shaped, rests against both sides of an outer circumference of the ball bearing (34a – 34e) and is non-rotatably connected to the drive shaft (16a – 16e).

27. (previously presented) The eccentric transmission as recited in claim 1, wherein the drive shaft (16a – 16e) is supported in a housing of the hand-held power tool (18a – 18e) by a ball bearing (46a – 46e).